

MATERIALE DI SUPPORTO TURBOMACCHINE

Equazione di continuità (conservazione della massa)

$$\dot{m} = \rho C_n A = \text{cost}$$

Triangoli delle velocità

$$\vec{C} = \vec{U} + \vec{W}$$

Equazione di Eulero (conservazione del momento angolare)

$$T = \dot{m}(r_2 C_{u2} - r_1 C_{u1})$$

$$P = T\omega = \dot{m}(U_2 C_{u2} - U_1 C_{u1})$$

Primo principio della termodinamica (conservazione dell'energia)

$$P[W] = \dot{m} \left[\frac{kg}{s} \right] L \left[\frac{J}{kg} \right] \quad L = \text{lavoro specifico}$$

$$P = \dot{m}(h_2^0 - h_1^0) = \dot{m}(h_2 + C_2^2/2 - h_1 - C_1^2/2)$$

Risultati utili

$$L = L_{az} + L_{reaz} = C_2^2/2 - C_1^2/2 + U_2^2/2 - U_1^2/2 + W_1^2/2 - W_2^2/2$$

$$L_{az} = C_2^2/2 - C_1^2/2$$

$$L_{reaz} = U_2^2/2 - U_1^2/2 + W_1^2/2 - W_2^2/2$$

$$h_2 - h_1 = U_2^2/2 - U_1^2/2 + W_1^2/2 - W_2^2/2$$

$$h_{2rel}^0 - h_{1rel}^0 = U_2^2/2 - U_1^2/2$$

$$h_2^0 - h_1^0 = U_2 C_{u2} - U_1 C_{u1}$$

Grado di reazione

$$R = \frac{L_{reaz}}{L} = \frac{h_2 - h_1}{h_2^0 - h_1^0}$$

Rotalpia

$$I = h + W^2/2 - U^2/2 = \text{cost}$$

$$I = h_{rel}^0 - U^2/2 = \text{cost}$$

$$I = h^0 - U C_u = \text{cost}$$

Trasformazione adiabatica reversibile (isentropica)

$$\delta Q = 0 \quad ds = \frac{\delta Q}{T} + ds_{irr} = 0 \quad L_{is} = \int_1^{2is} v dp \quad \frac{p}{\rho^k} = cost$$

Trasformazione adiabatica irreversibile (reale)

$$\delta Q = 0 \quad ds = ds_{irr} \quad L_{irr} > L_{is} \quad \text{compressore}$$

$$\delta Q = 0 \quad ds = ds_{irr} \quad L_{irr} < L_{is} \quad \text{turbina}$$

Trasformazione reversibile politropica (equivalente)

$$\delta Q = T ds \quad ds_{irr} = 0 \quad L_{irr} > L_{pol} = \int_1^2 v dp > L_{is} \quad \frac{p}{\rho^n} = cost \quad n > k \quad \text{compressore}$$

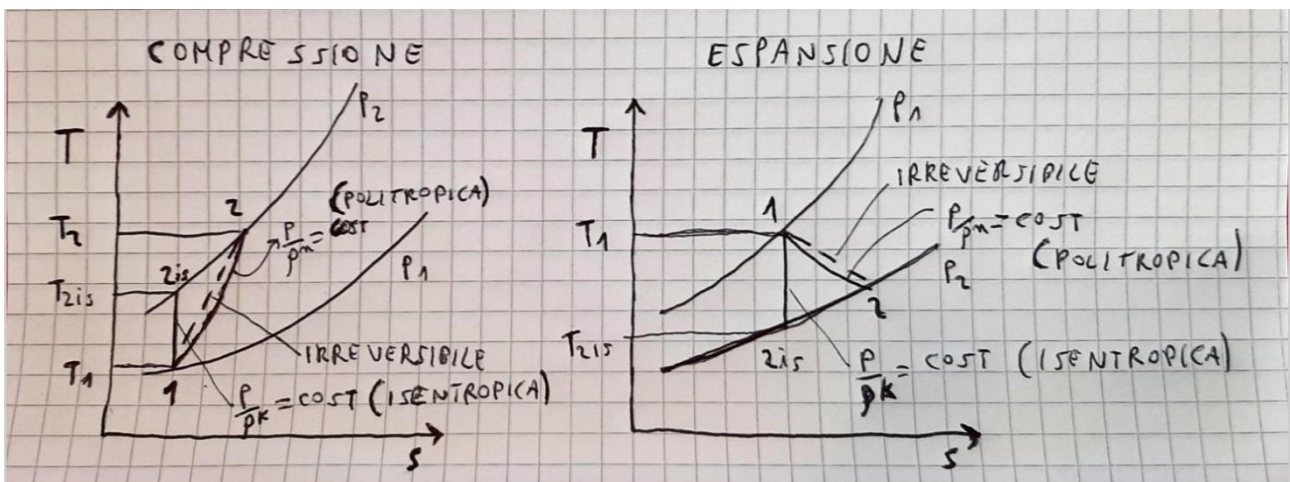
$$\delta Q = T ds \quad ds_{irr} = 0 \quad L_{irr} < L_{is} < L_{pol} = \int_1^2 v dp \quad \frac{p}{\rho^n} = cost \quad n < k \quad \text{turbina}$$

$$\frac{n-1}{n} = \frac{k-1}{k \eta_{pol}} \quad \Leftrightarrow \quad \eta_{pol} = \frac{k-1}{k} \frac{n}{n-1} \quad \text{compressore}$$

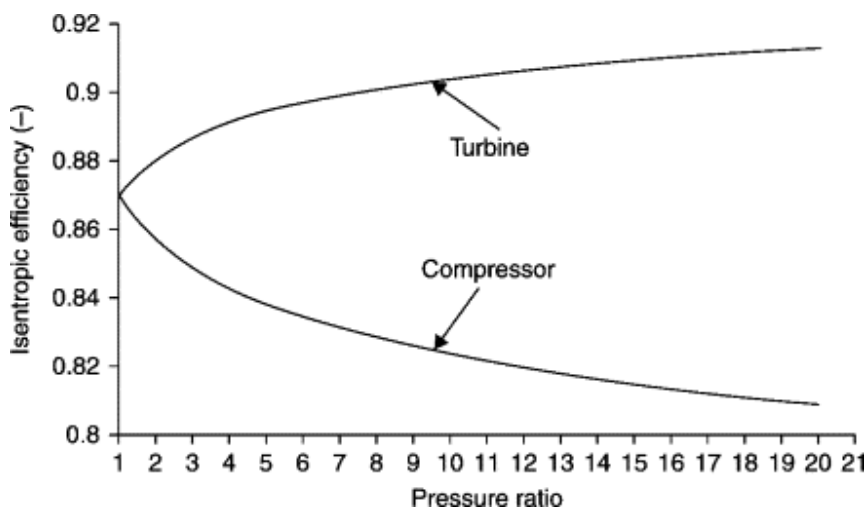
$$\frac{n-1}{n} = \frac{k-1}{k} \eta_{pol} \quad \Leftrightarrow \quad \eta_{pol} = \frac{n-1}{n} \frac{k}{k-1} \quad \text{turbina}$$

$$\eta_{pol} = \frac{L_{pol}}{L_{irr}} > \eta_{is} = \frac{L_{is}}{L_{irr}} \quad \eta_{is} = \frac{(\pi_c^{(k-1)/k} - 1)}{(\pi_c^{(k-1)/k \eta_{pol}} - 1)} \quad \text{compressore}$$

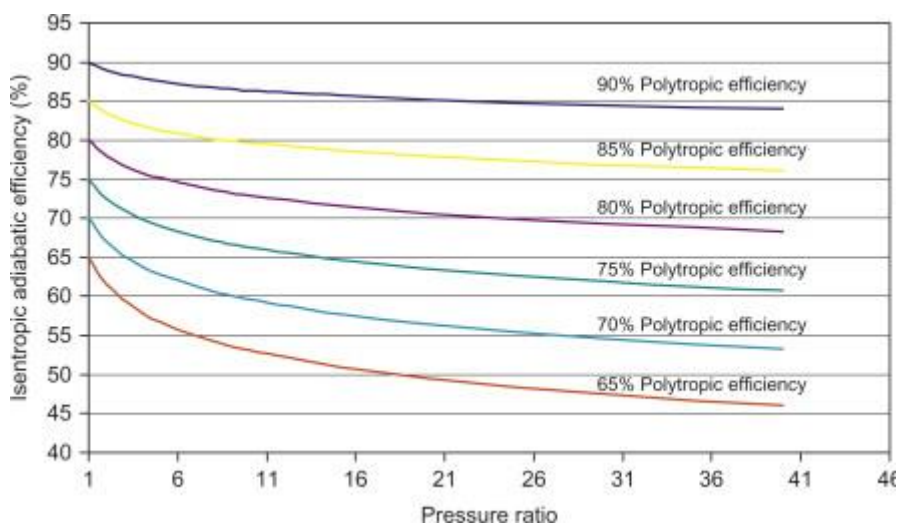
$$\eta_{pol} = \frac{L_{irr}}{L_{pol}} < \eta_{is} = \frac{L_{irr}}{L_{is}} \quad \eta_{is} = \frac{(1 - \pi_e^{\eta_{pol}(k-1)/k})}{(1 - \pi_e^{(k-1)/k})} \quad \text{turbina}$$



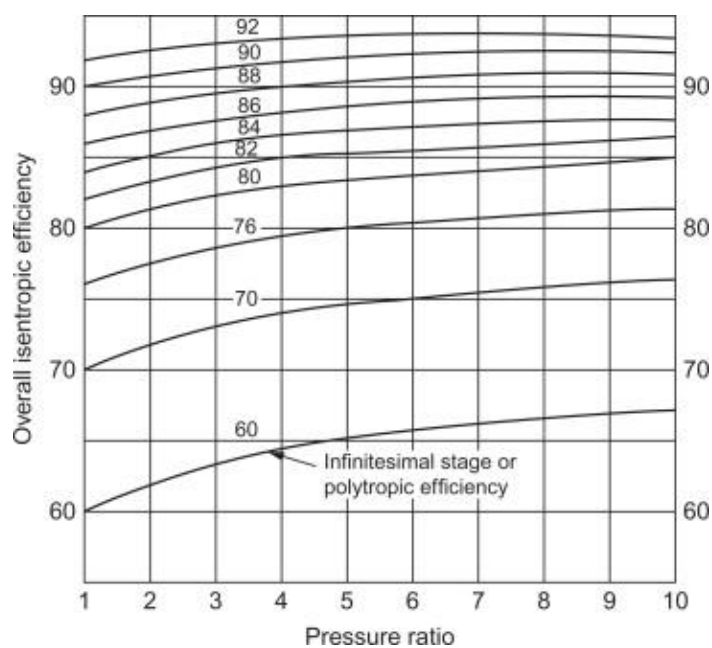
Confronto tra trasformazione isentropica, politropica reversibile e adiabatica irreversibile in compressione (sinistra) ed espansione (destra)



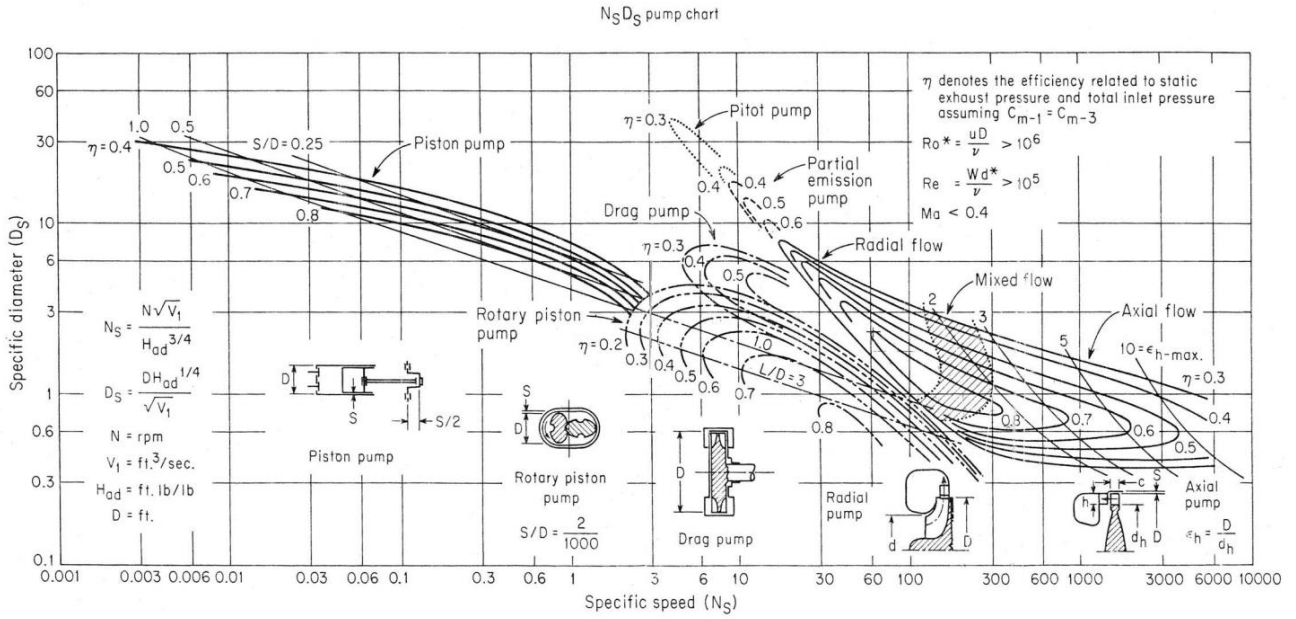
Relazione rendimento politropico vs rendimento isentropico



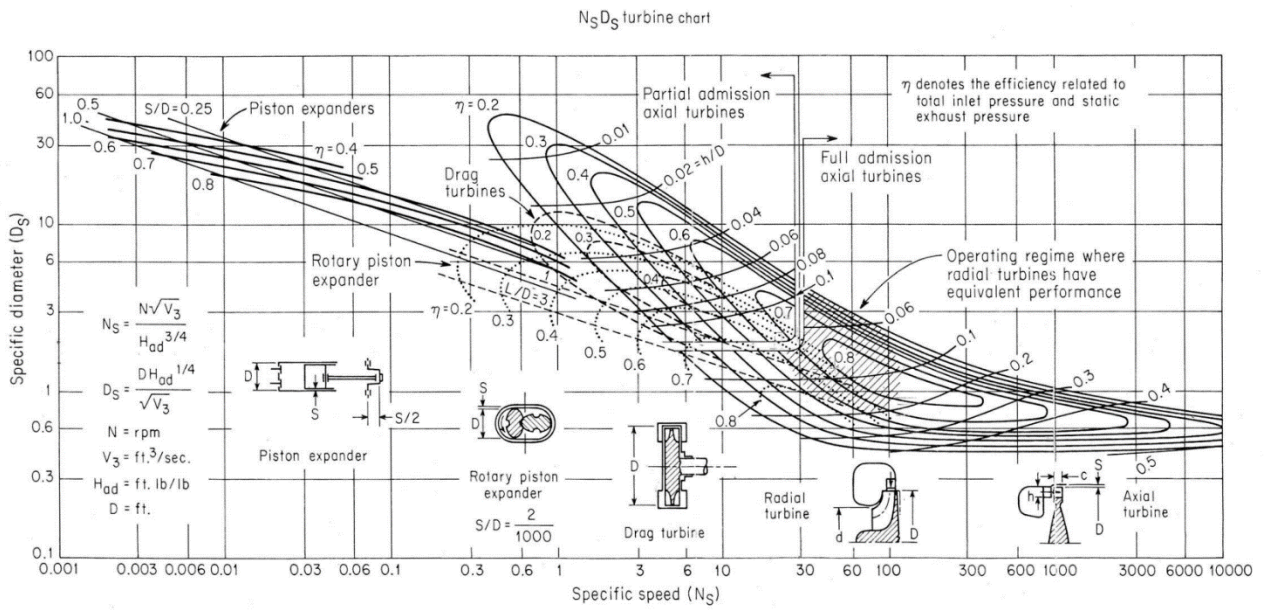
Relazione rendimento politropico vs rendimento isentropico in compressione



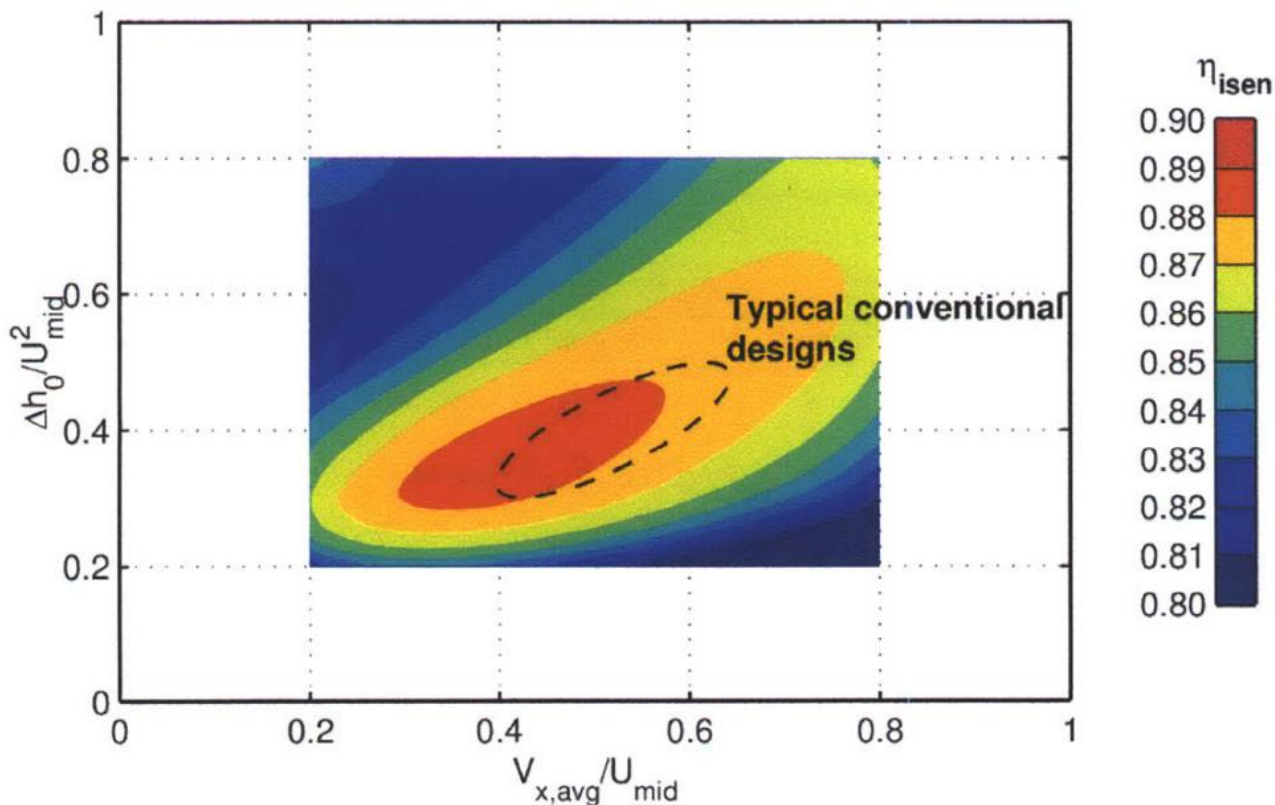
Relazione rendimento politropico vs rendimento isentropico in espansione



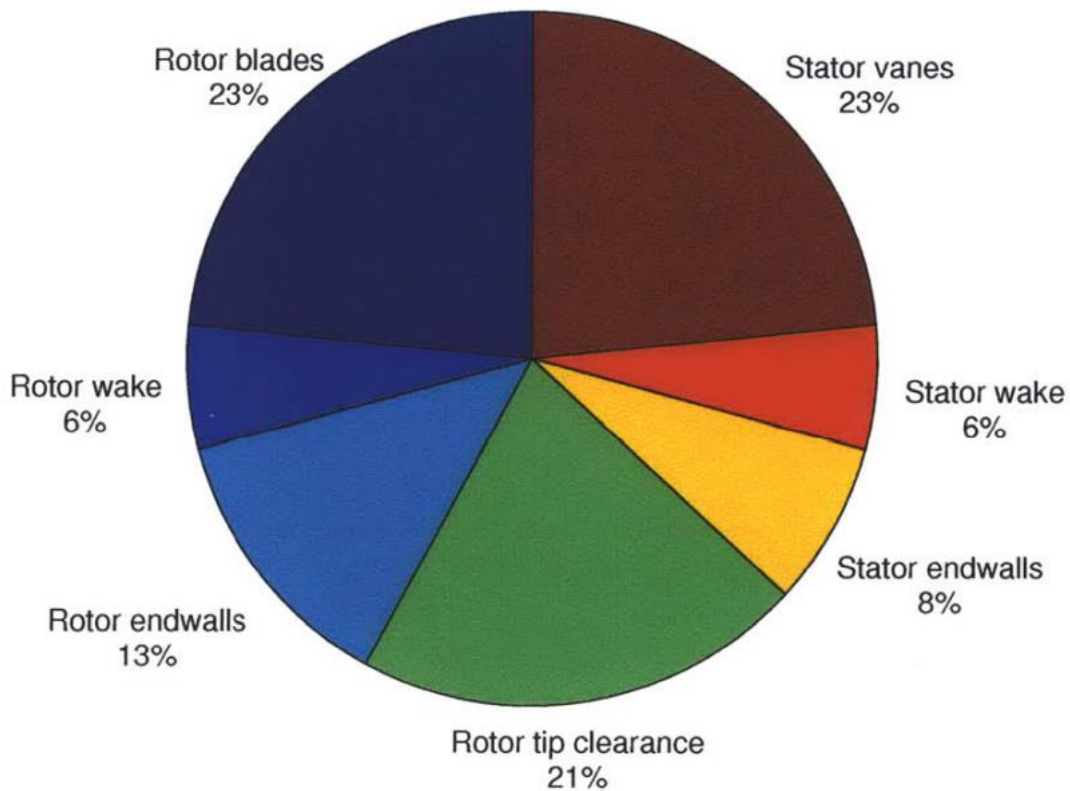
Pump selection chart



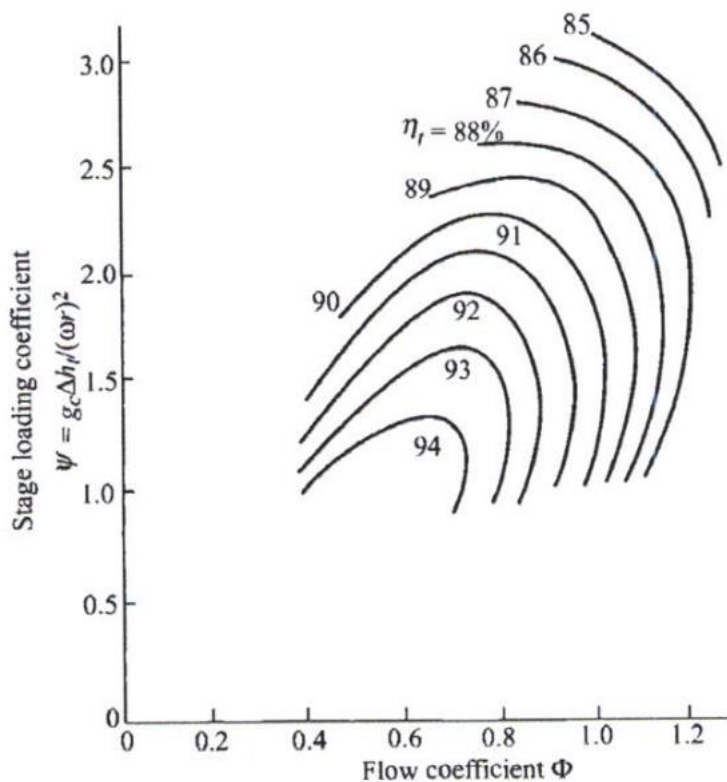
Turbine selection chart



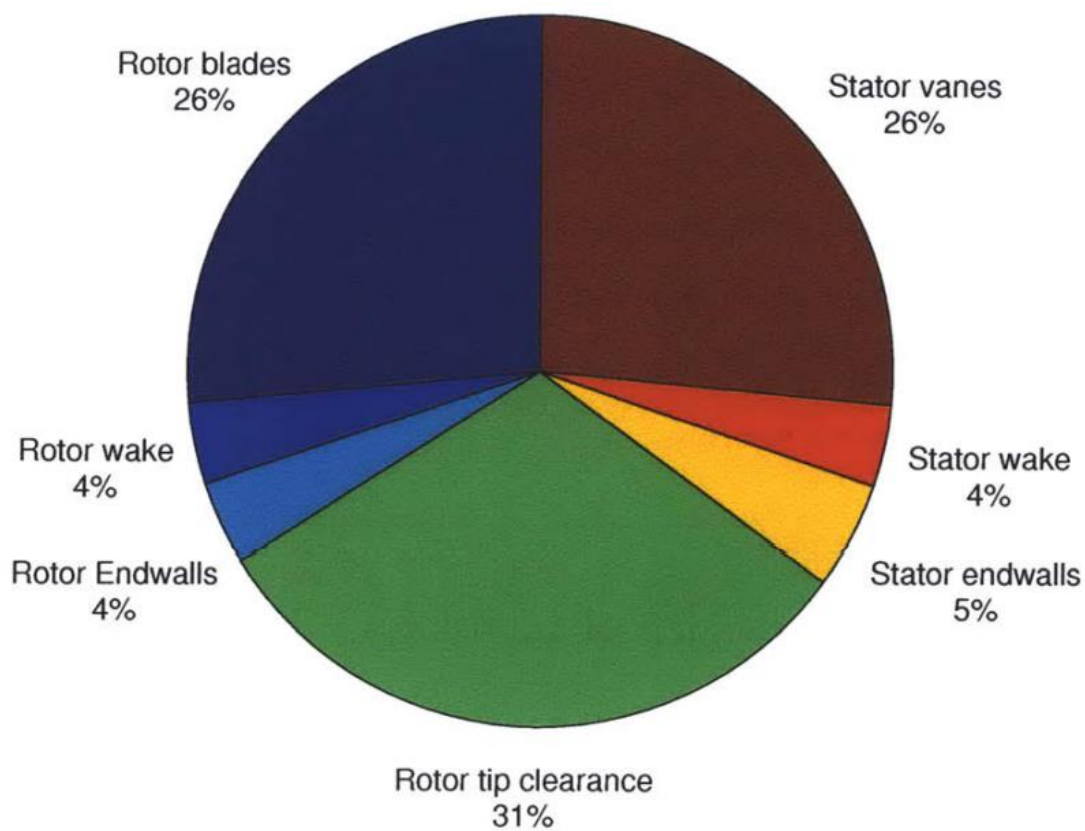
Smith chart, one-dimensional correlation of Wright & Miller, reproduced from Dickens & Day



Breakdown of losses at peak efficiency for a compressor stage



Turbine Smith chart based on experimental data, as produced in Mattingly



Breakdown of losses at peak efficiency for an uncooled turbine stage